

A SMART LEADING EDGE ARCHITECTURE AIMED AT PRESERVING LAMINAR REGIME

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Contents (1/2)

➤ Motivation of the work

- traditional high lift devices → pros & cons
- an innovative approach → the '*morphing*'
- objective of the work: droop nose

➤ Strategy definition

- internal compliant mechanism
- two working modalities: '*self adaptive*' and '*motor aided*'
- '*self adaptive*' working flow chart

➤ Design and modeling

- FE approach, non linear analysis for 1:6 and real sized model
- architecture optimization
- spring identification for '*self adaptive*' modality
- actuator identification for '*aided motor*' modality



Contents (2/2)

➤ Morphing performance

- 'droop nose' geometry parameters: vertical displacement, LE radius increase
- shape keeping under aerodynamic loads

➤ Conclusions

- summary of the activities
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- prototyping
- experimental characterization



Motivation of the work (1/3)

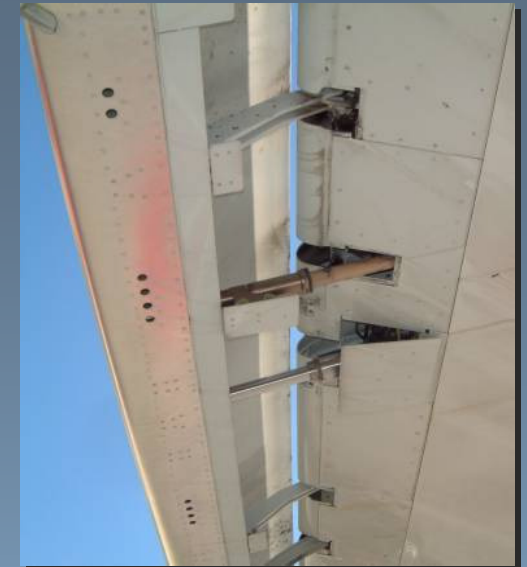
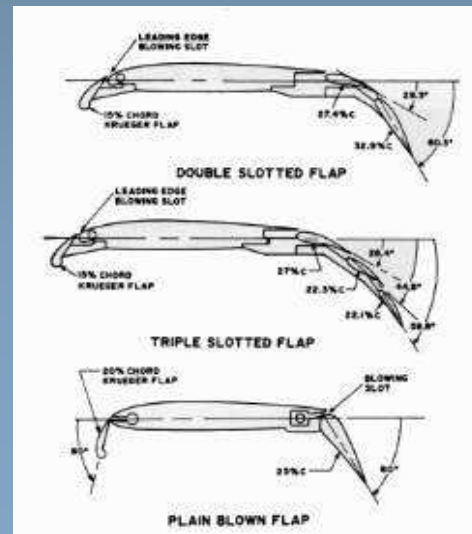
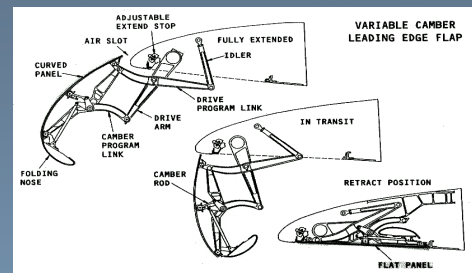
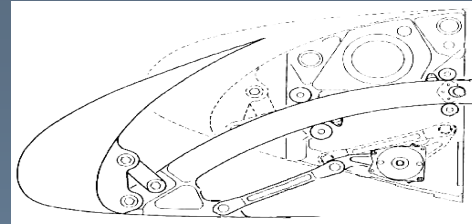
Traditional high lift devices → pros & cons

➤ Pros:

- curvature, chord increase
- flow energisation through the slots
- consequent High Lift improvement

➤ Cons:

- heavy actuation mechanisms (up to 50% of wing total weight)
- gaps induced discontinuities even in cruise condition
- Turbulence priming



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An innovative approach → the ‘morphing’

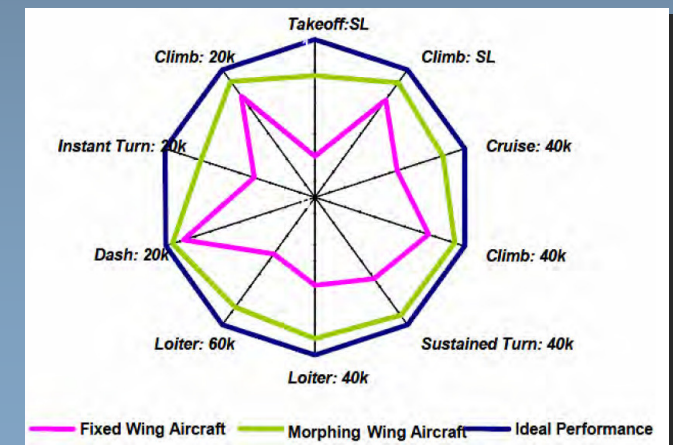
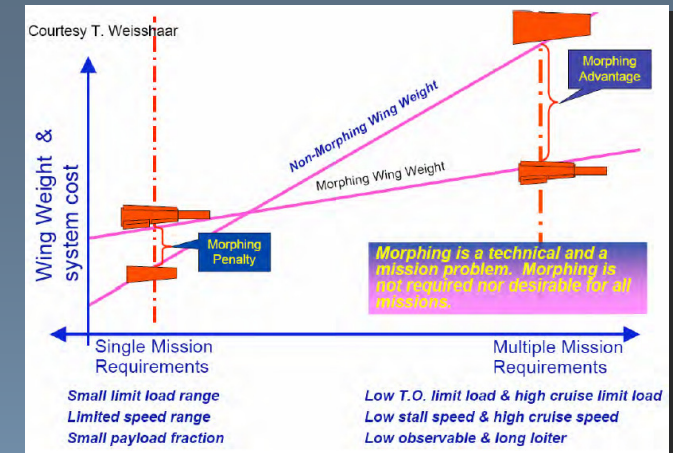
➤ Morphing:

- $\mu\omicron\rho\phi\acute{\eta}$, $-\tilde{\eta}\zeta$, $\acute{\eta}$: shape
- smooth and at same time large variations of geometry
- a paradox: the structure must morph (→ flexibility) and withstand aerodynamic loads (→ rigidity)
- technical solutions: ‘Smart Structure and Material’ concept

➤ Advantages in aeronautics:

- adaptability → versatility to a wide variety of flight conditions
- multi-point → accommodation of even contradictory mission scenarios
- efficient → improvement of weight, volume, costs...

Motivation of the work (2/3)



Objective of the work

- Design of a 'Smart Leading Edge' (SLE) architecture for:
 - High Lift generation
 - Laminar regime preserving for different flight conditions (cruise, take-off and landing)

- Innovation:
 - **self-adaptive under external loads...**



Strategy definition (1/3)

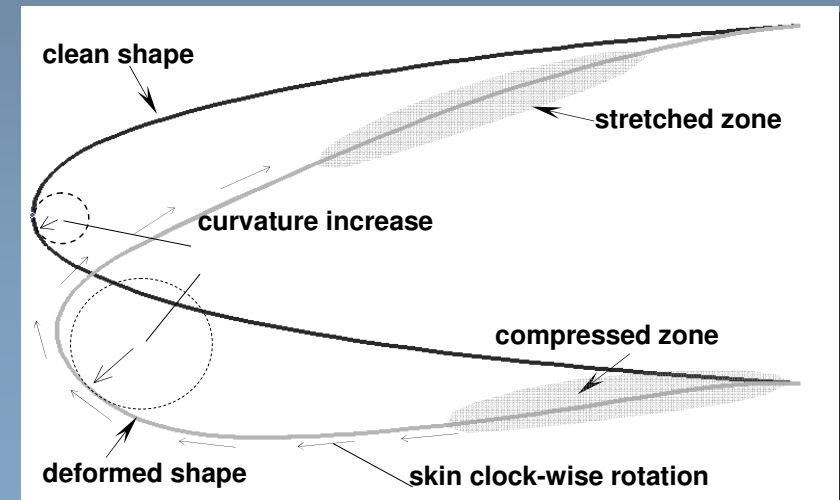
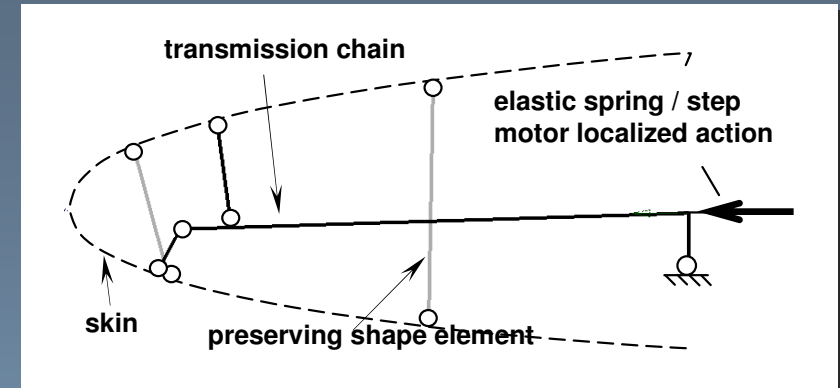
Internal compliant mechanism

➤ The SLE is constituted by:

- a skin element → transmission of external load to the structure
- an internal mechanism → actuation of deformation

➤ Internal cinematic chain design criteria:

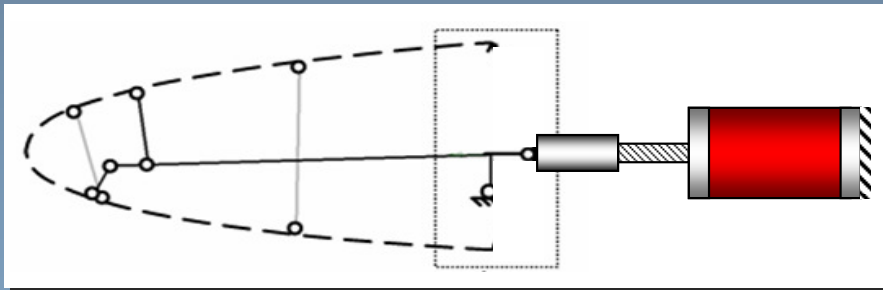
- transmission of max vertical displacement
- leading edge radius increase
- no collapse or instability within the skin
- aerodynamic shape maintenance



Two working modalities

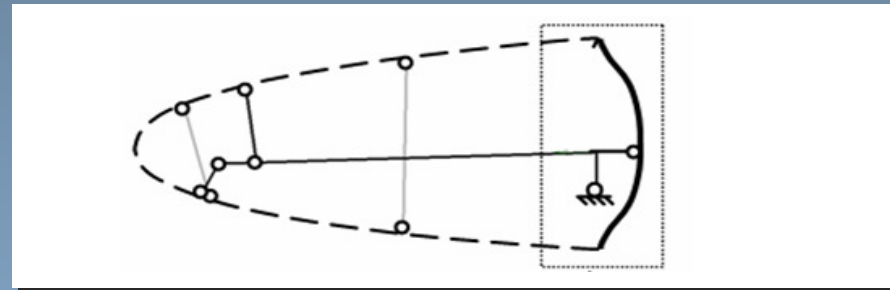
'Motor aided' configuration:

- the deformation is caused by an actuator (located in the wing box available space)
- actuator performance must guarantee deformed shape under external loads

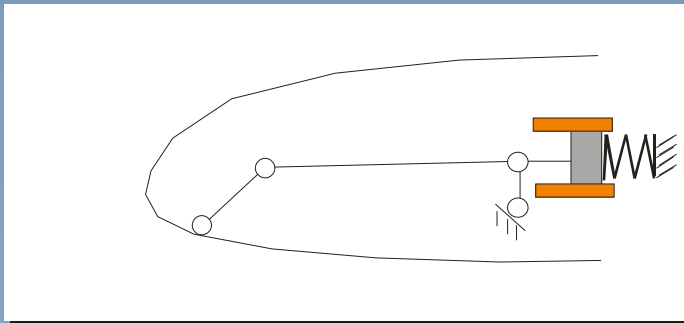
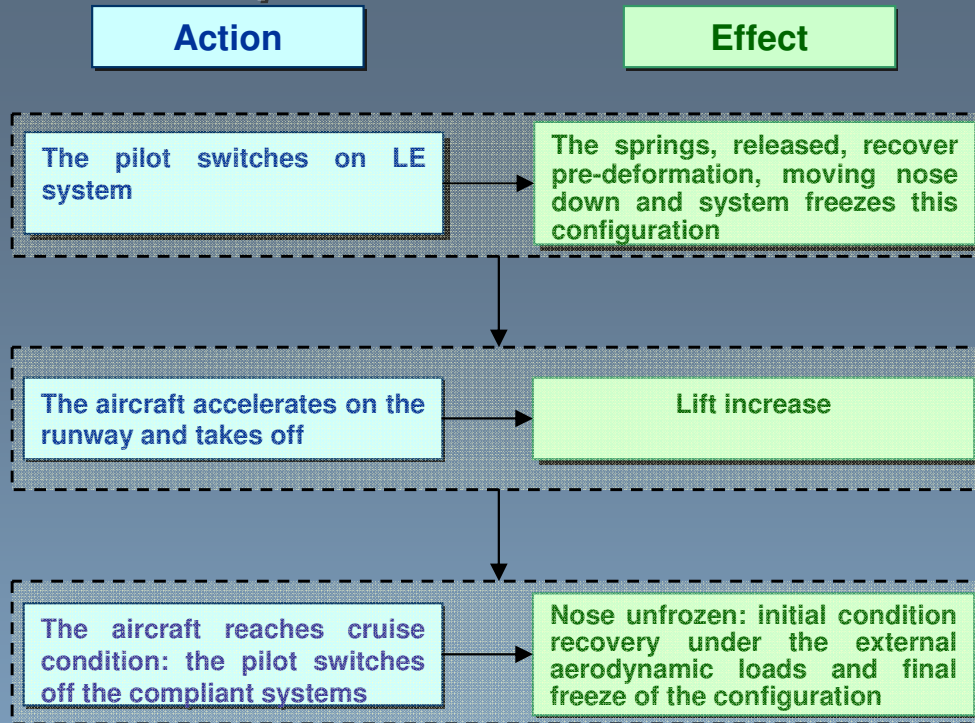


'Self-adaptive' configuration:

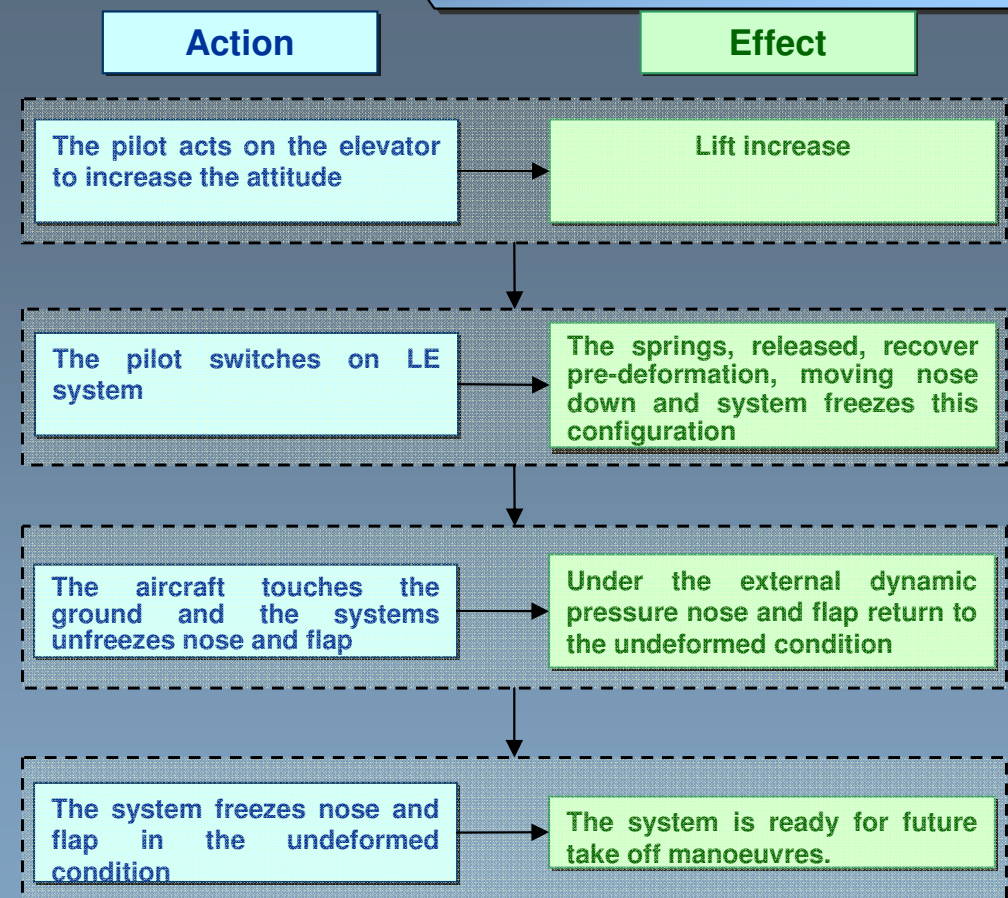
- the deformation is caused by a variable stiffness spring (locked and unlocked by a brake)
- spring rigidity is tuned on expected aerodynamic loads



'Self adaptive' flow chart



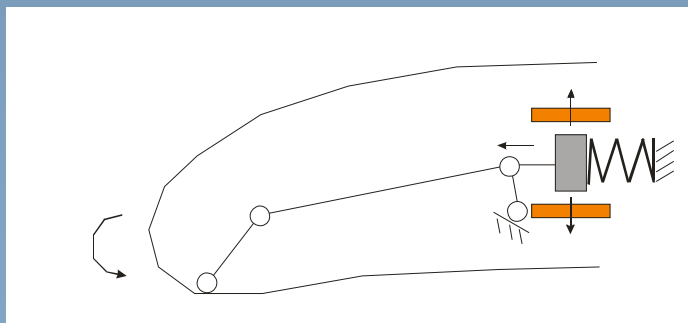
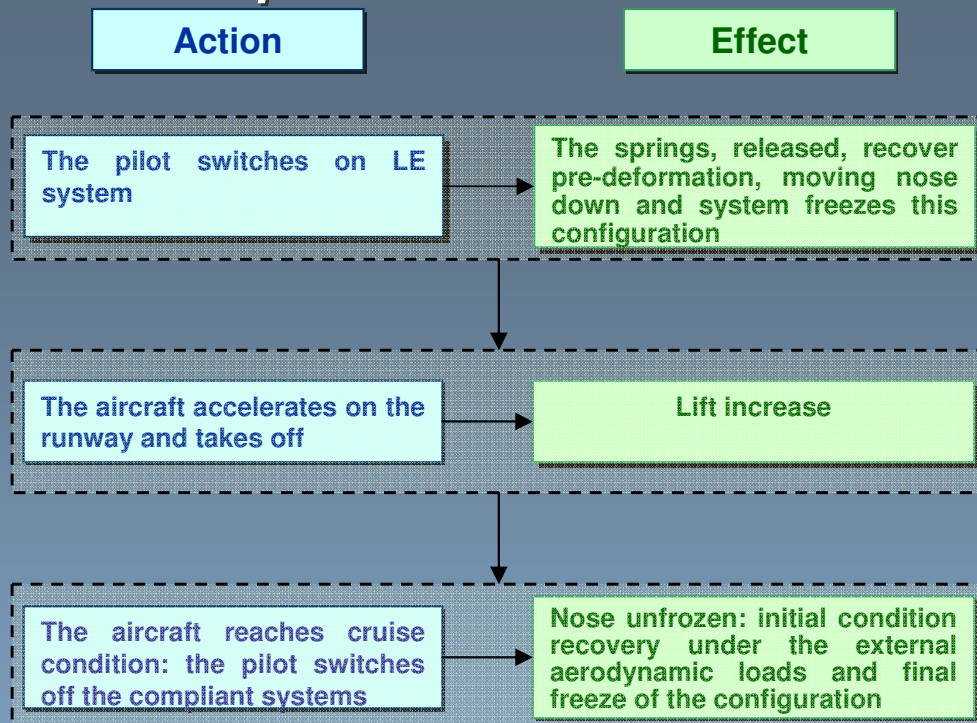
Strategy definition (3/3)



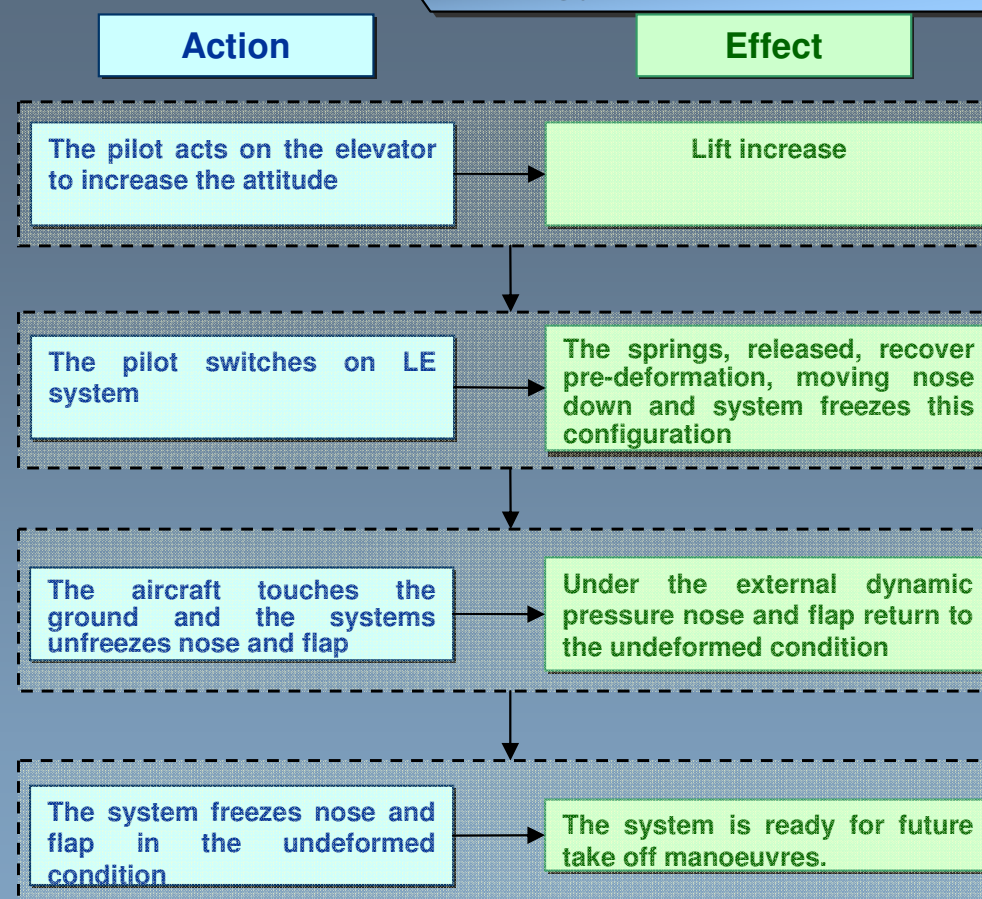
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'Self adaptive' flow chart



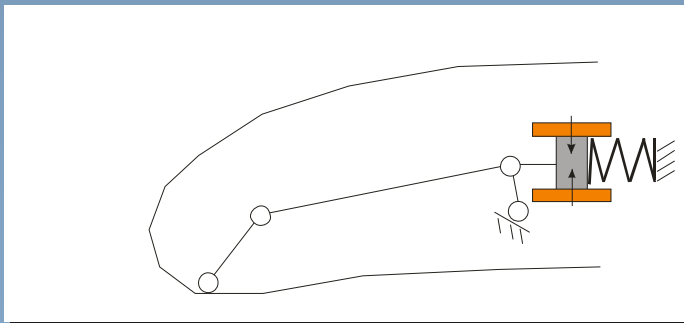
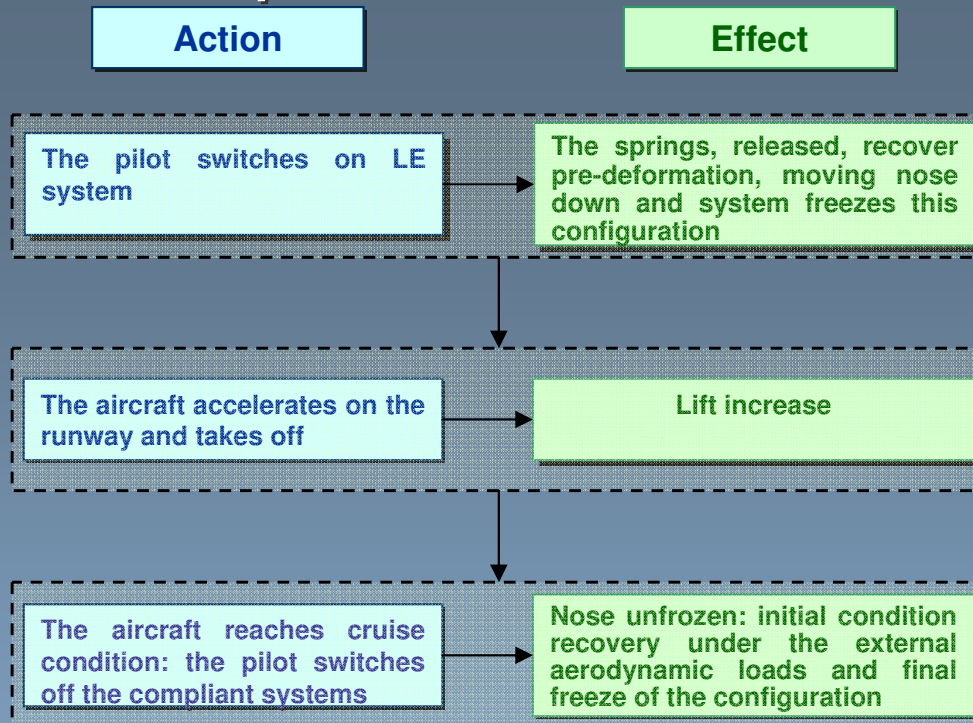
Strategy definition (3/3)



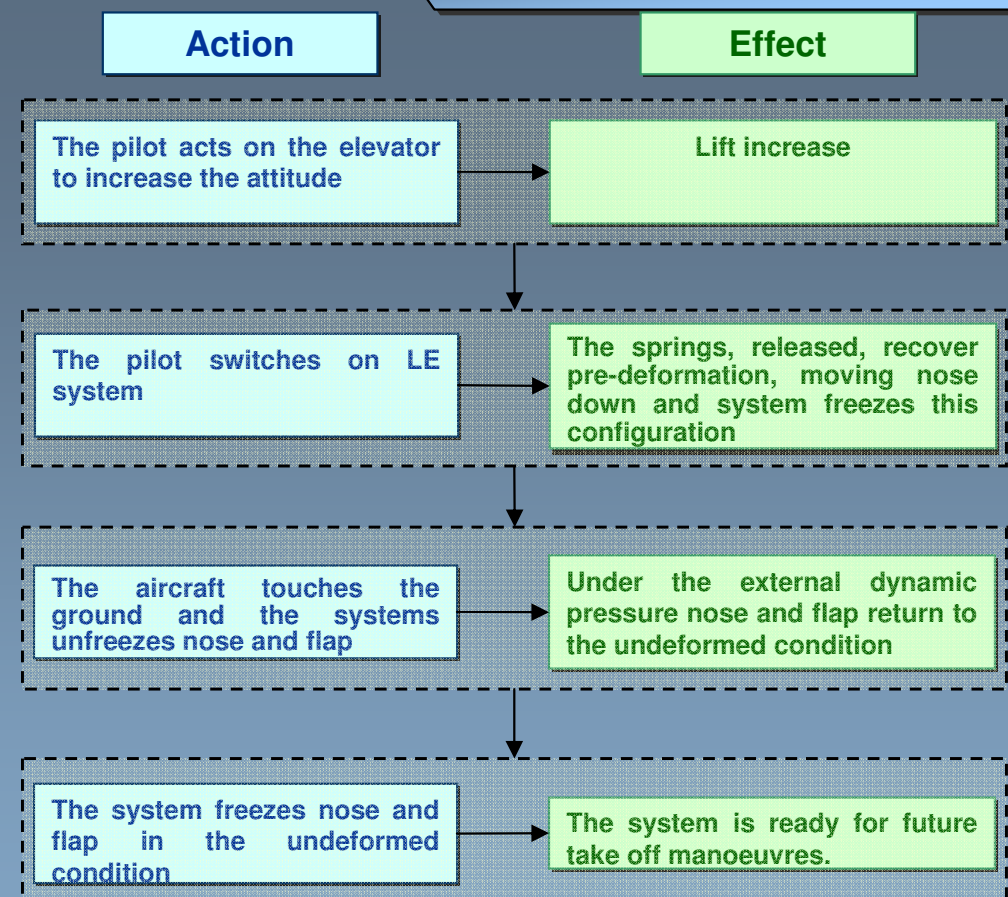
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'Self adaptive' flow chart



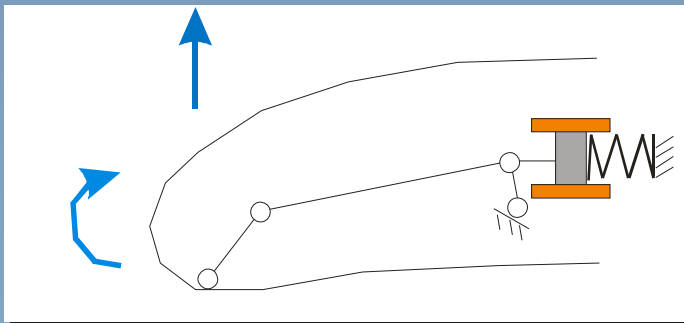
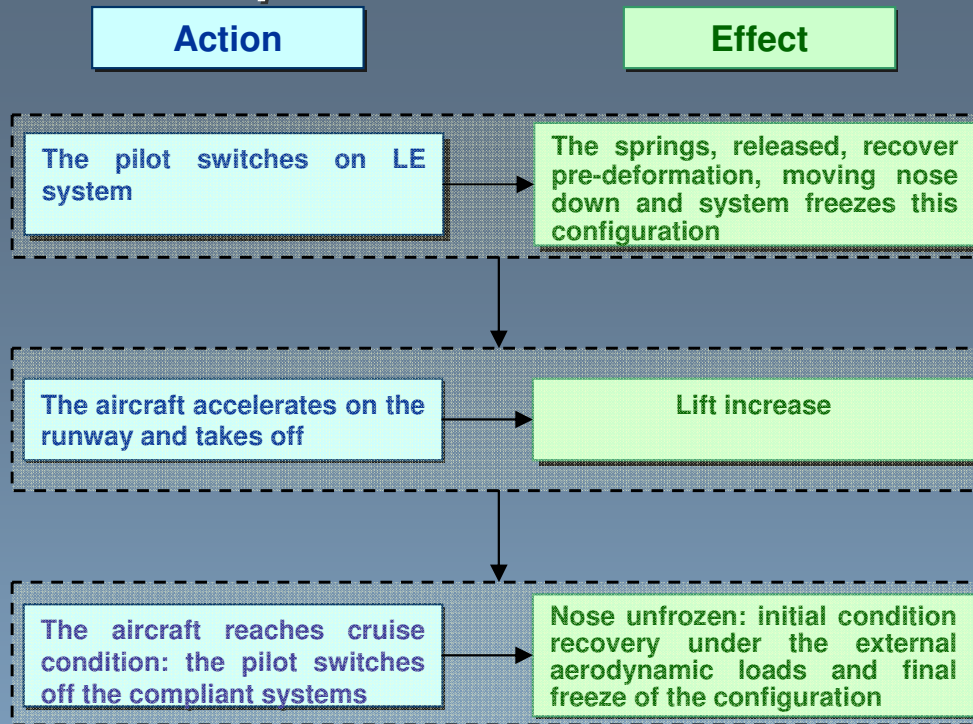
Strategy definition (3/3)



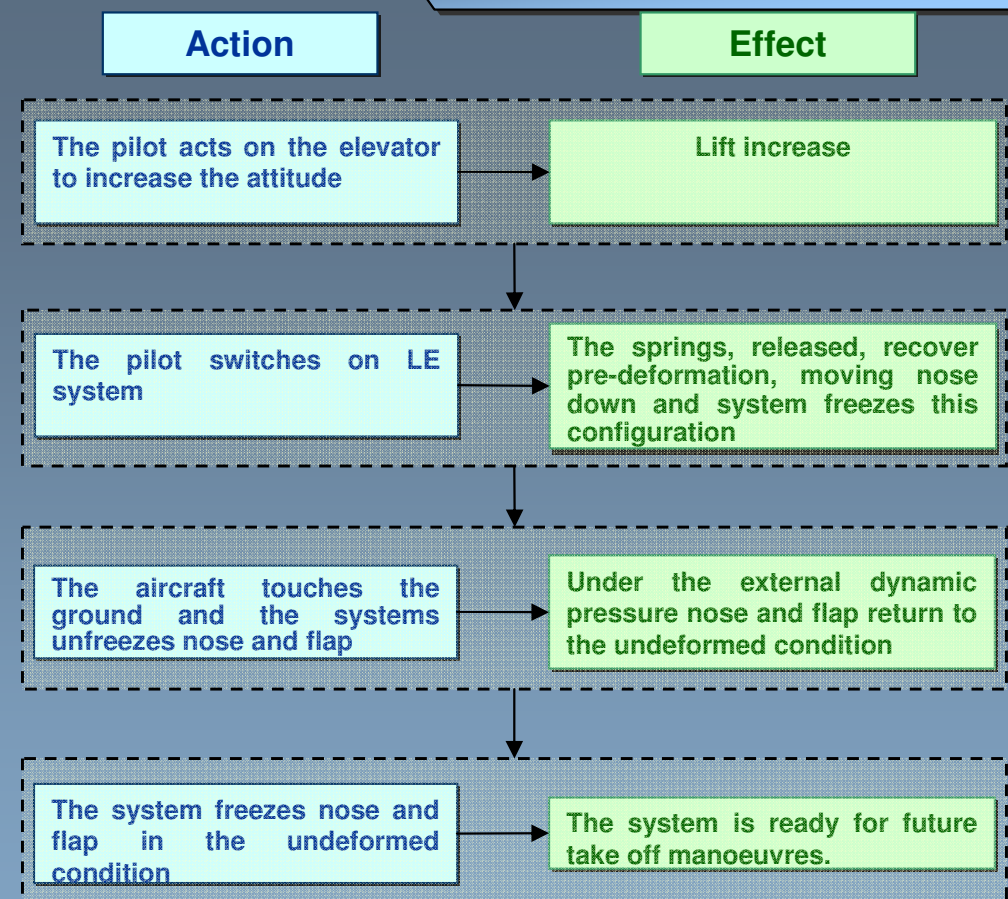
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'Self adaptive' flow chart



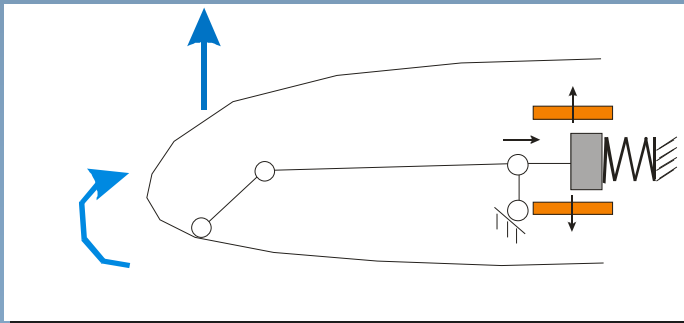
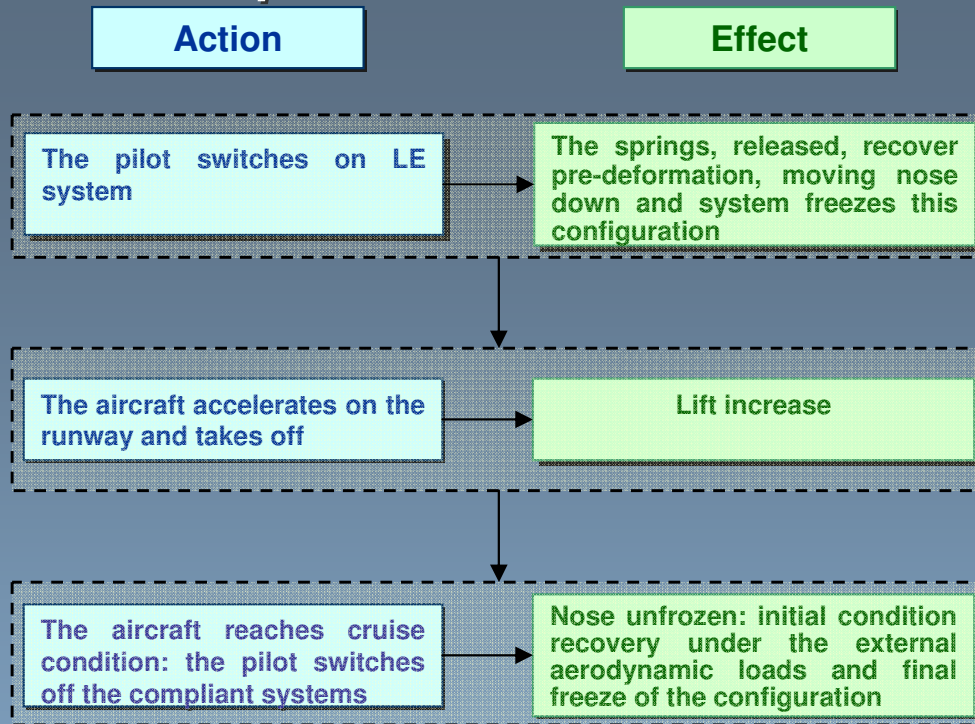
Strategy definition (3/3)



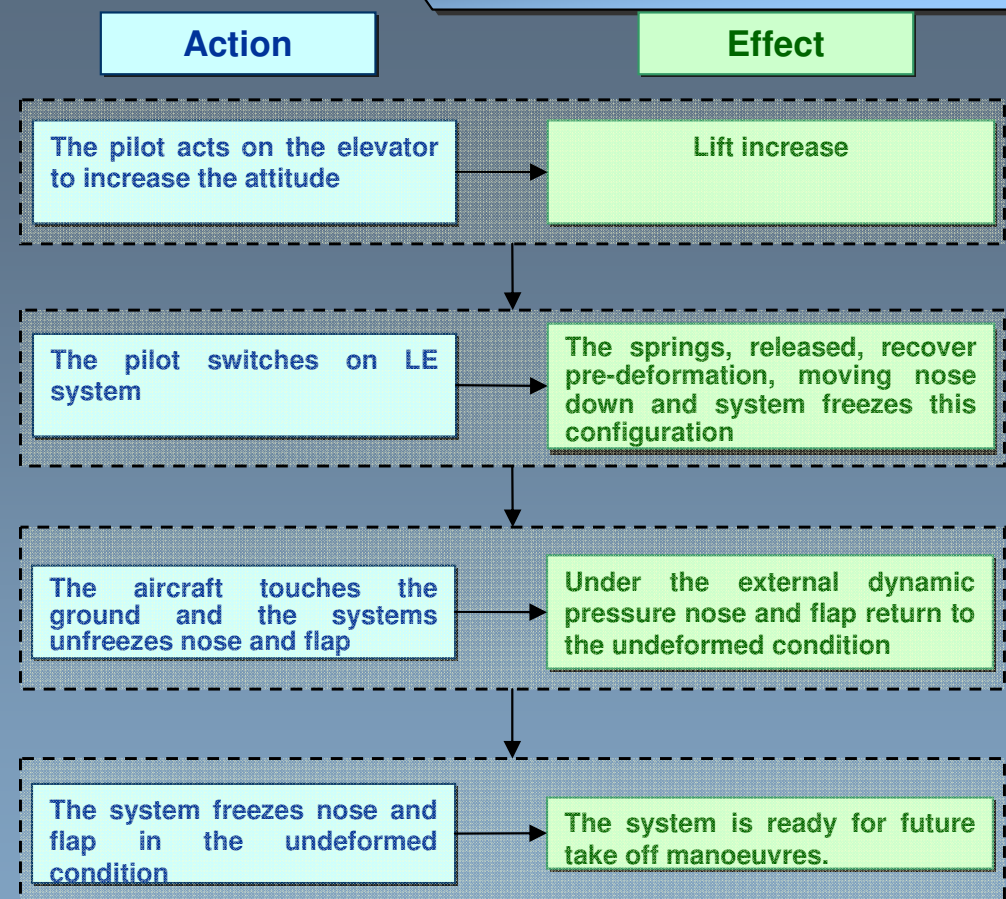
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'Self adaptive' flow chart



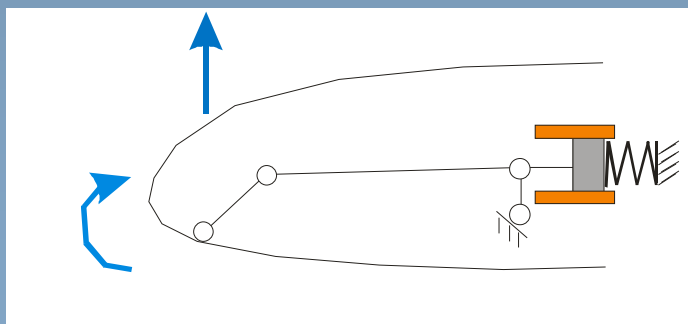
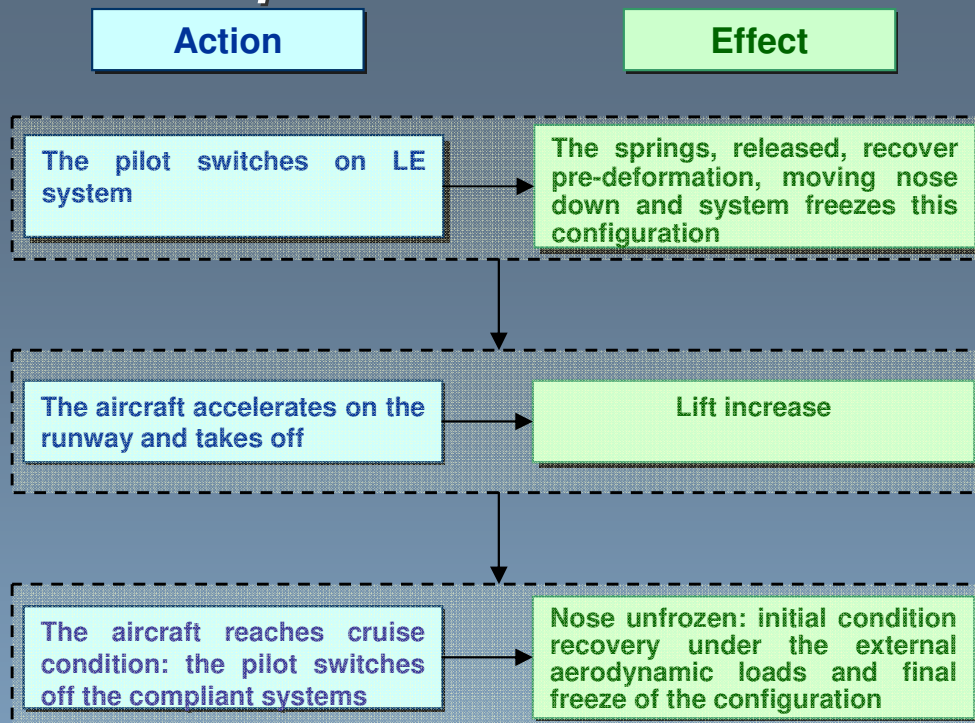
Strategy definition (3/3)



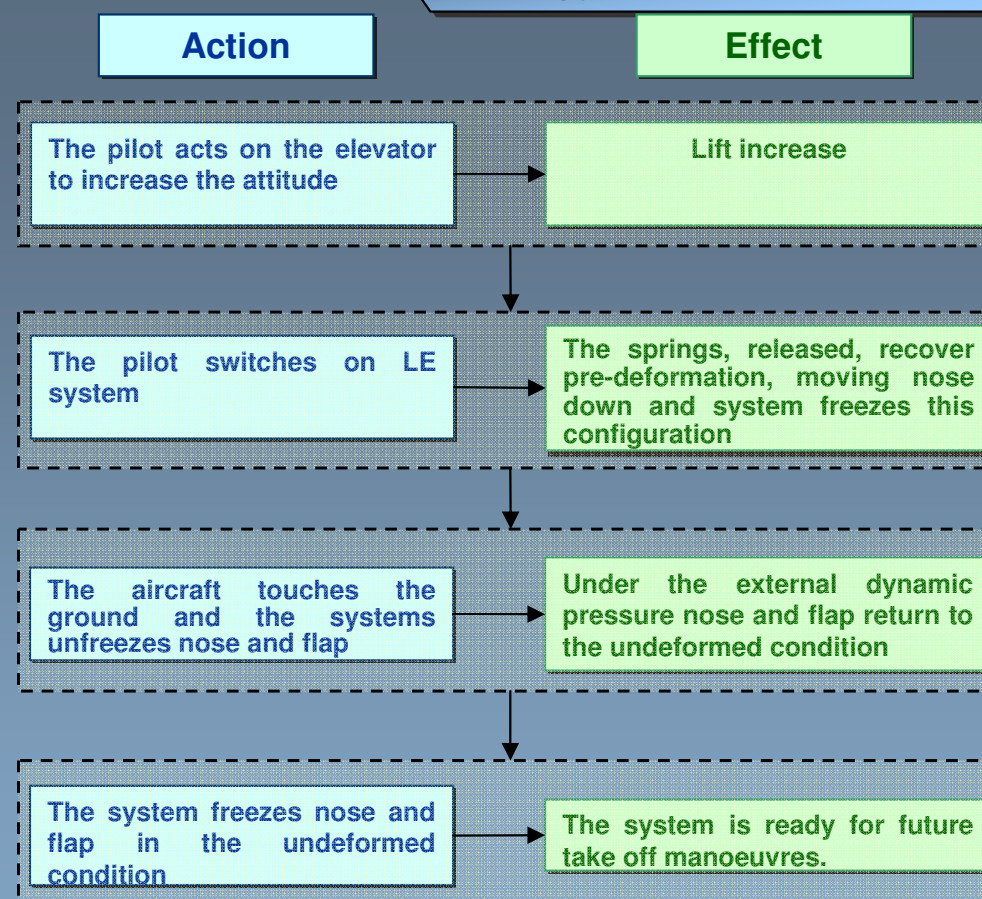
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'Self adaptive' flow chart



Strategy definition (3/3)



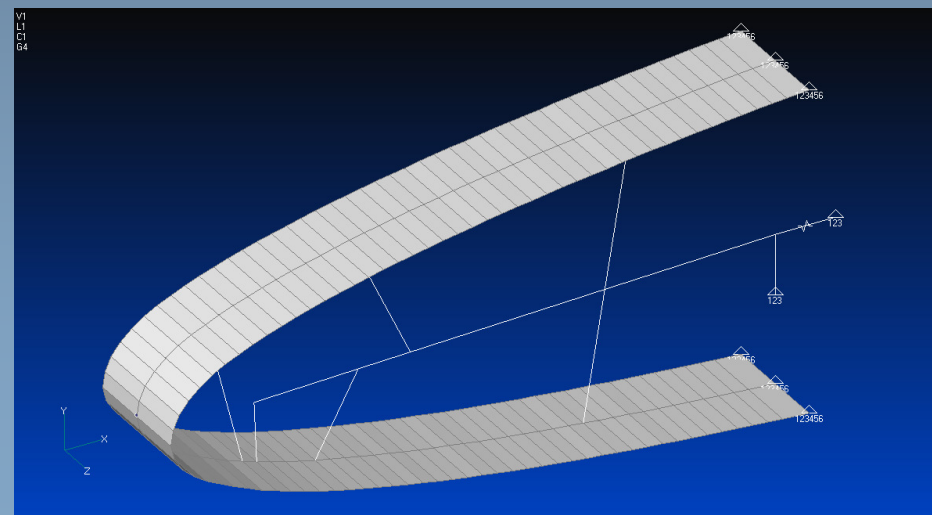
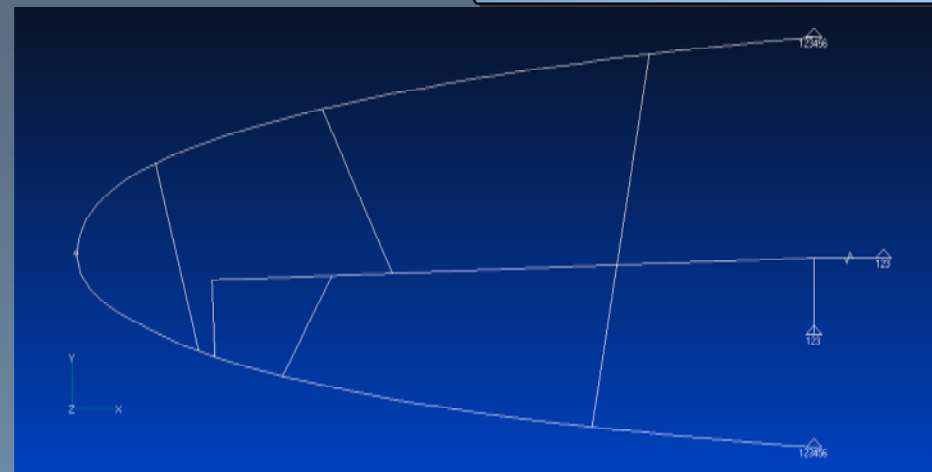
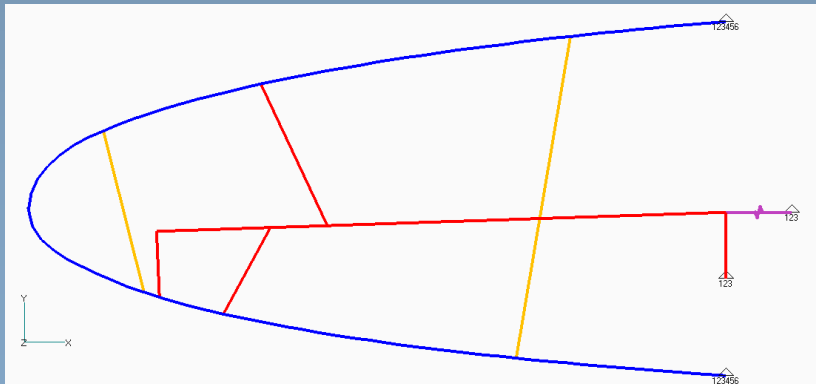
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Finite Element Modeling (1:6 scaled)

Design and modeling (1/7)

FE entities	
Nodes number	283
Elements total number	195
Skin elements (MSC/Nastran CQUAD4)	178
Tie rods elements (MSC/Nastran CBEAM)	16
Spring element (MSC/Nastran CEALS2)	1
SLE geometry	
SLE chord	112 mm on a total chord of 500 mm
SLE span	20 mm
Skin features	
Thickness	0,25 mm
Materials: AISI 301 Stainless Steel	E=210 GPa $\nu=0,32$ $\sigma_u=800\text{MPa}$
Tie rods features	
Equivalent area	50,27 mm ²
Inertia moments	$I_1=I_2=201\text{mm}^4$
Material 7075 T6 Aluminium Alloy	E=70 GPa $\nu=0,32$ $\sigma_u=472\text{MPa}$



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Design and modeling (2/7)

Optimization

➤ Target:

- achieve preliminary aerodynamic requirements: vertical displacement, leading edge radius

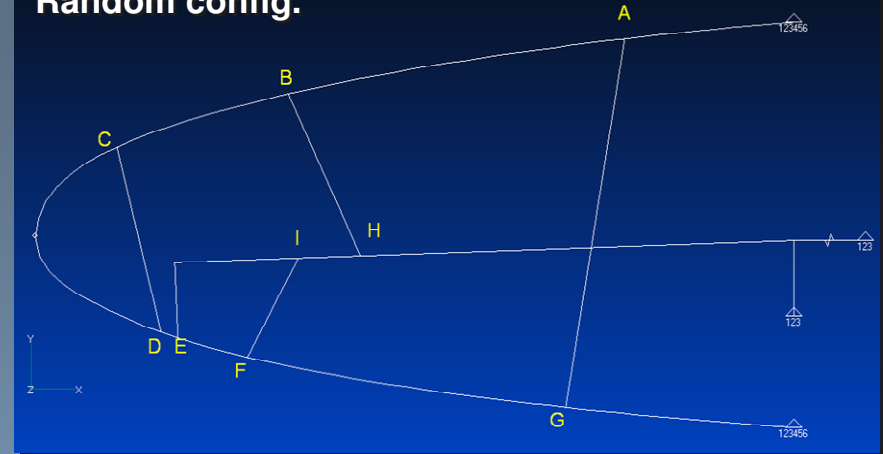
➤ Constraints:

- allowable stress not exceeded
- reasonable moving limits for rods extremes

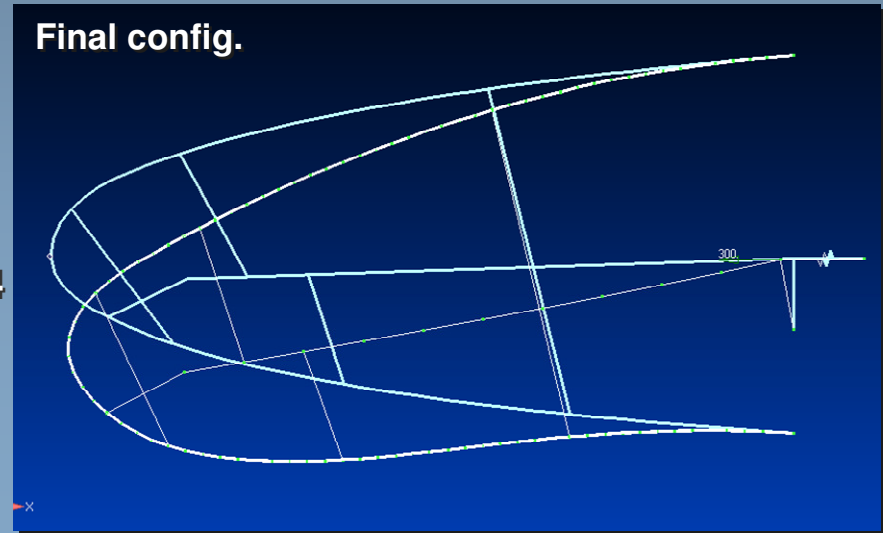
➤ Design variables:

- actuator force: 1
- rods extremes position (letters from "A" to "I"): 14
- parameters total number: 15

Random config.

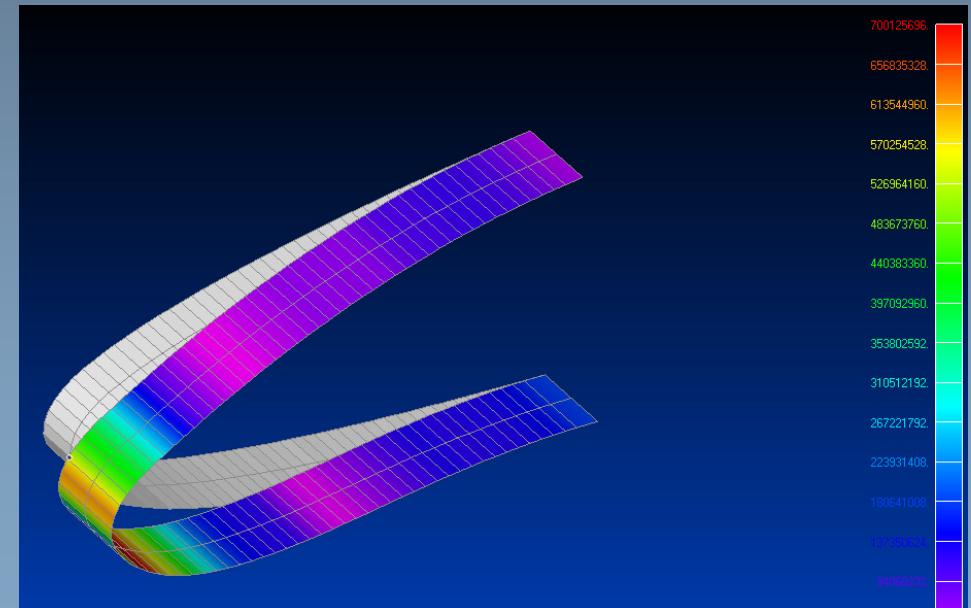
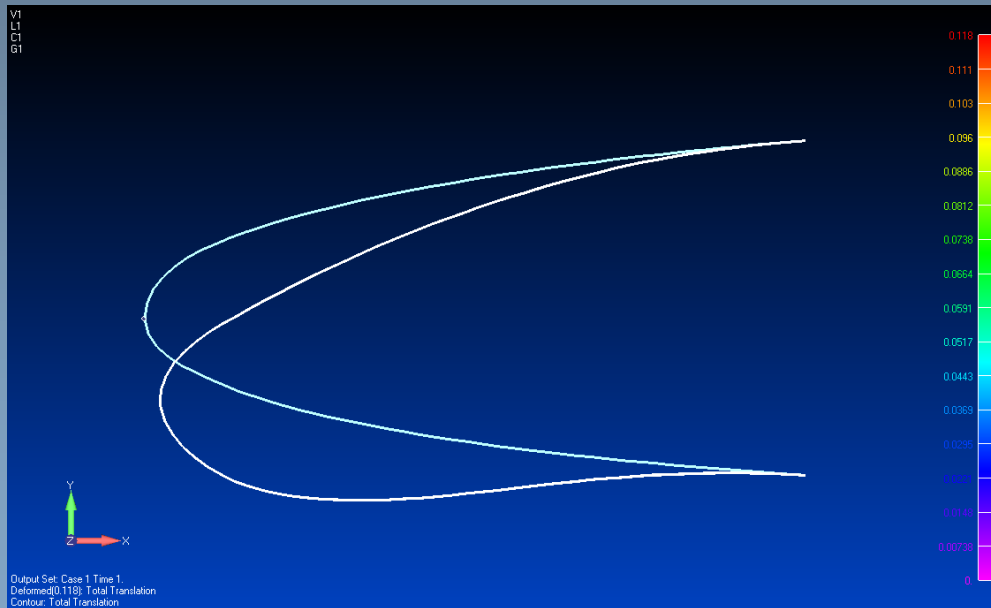


Final config.



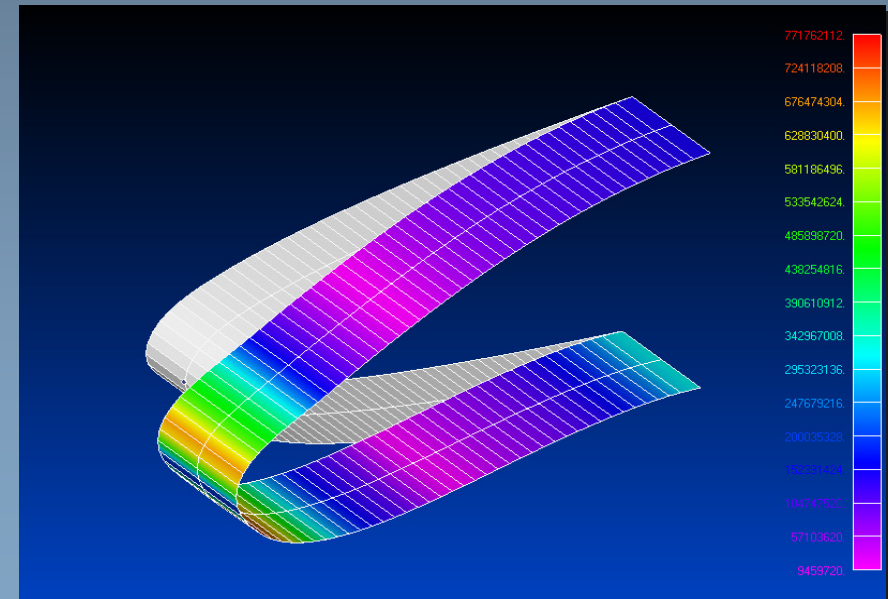
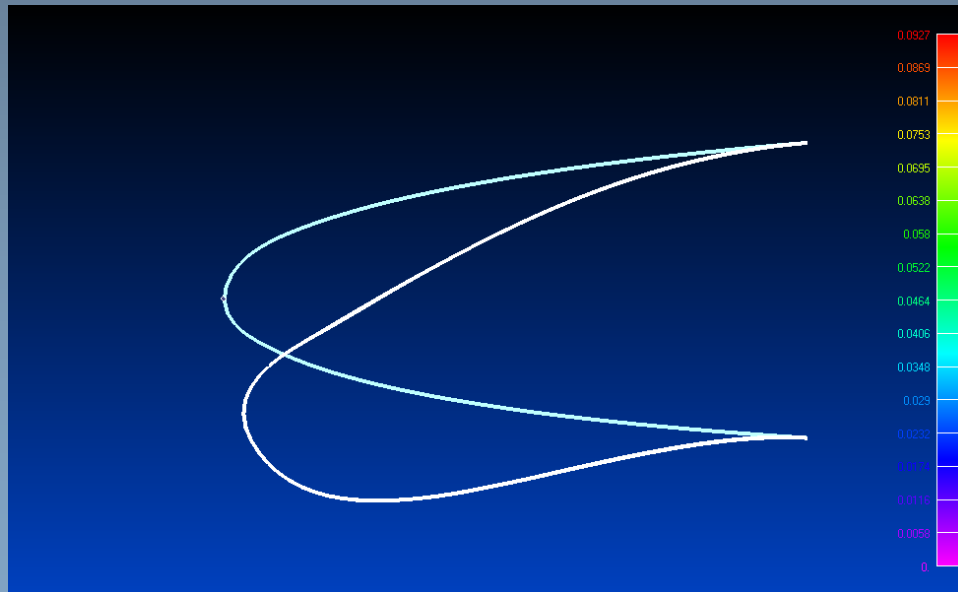
Finite Element Modeling (1:6 scaled, skin thickness: 0.25mm)

- max displacement, droop nose: 16 mm (14.3 % of the SLE chord, 3.2% of the total chord);
- max stress (within the skin): 700 MPa

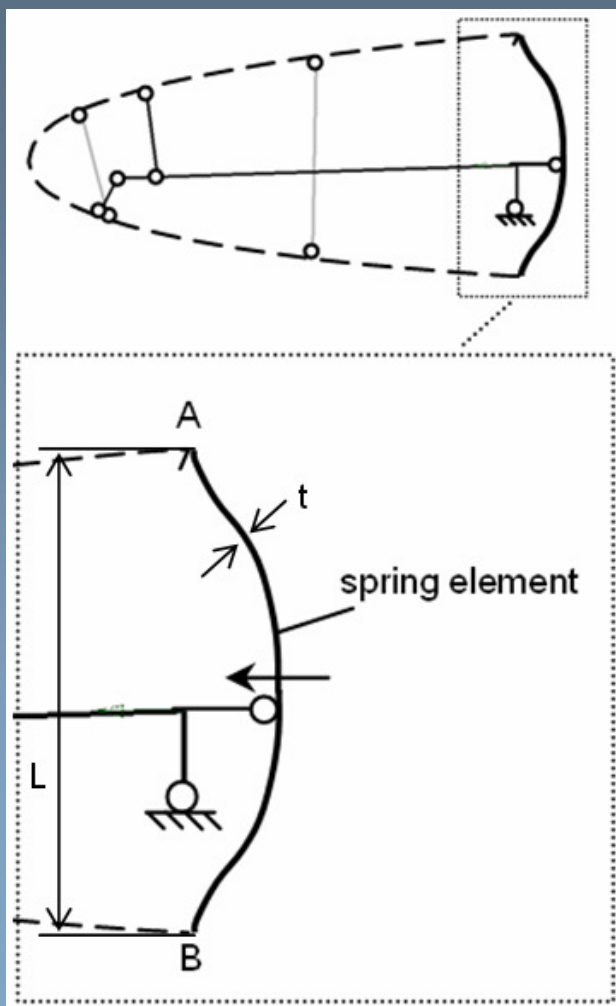


Finite Element Modeling (real size, skin thickness: 2.0 mm)

- max displacement, droop nose: 118 mm (13.2 % of the SLE chord, 3.1% of the total chord);
- max stress (within the skin): 700 MPa



Spring identification for 'self adaptive' modality



1:6 scaled	
Re	7,00E+06
M	0,6
V inf (m/s)	210
Applied force (N)	1700
Applied force per 1 mm span (N/mm)	85
Point displacement (m)	0,0020617
Elastic constant (N/m per mm span)	41228

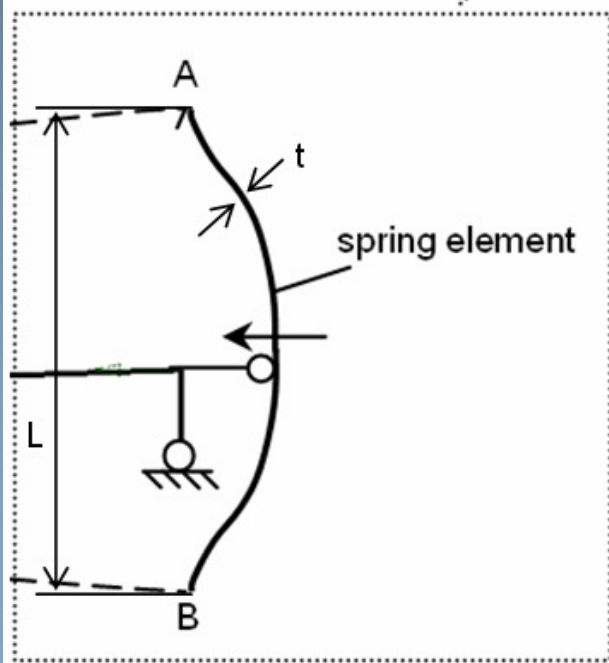
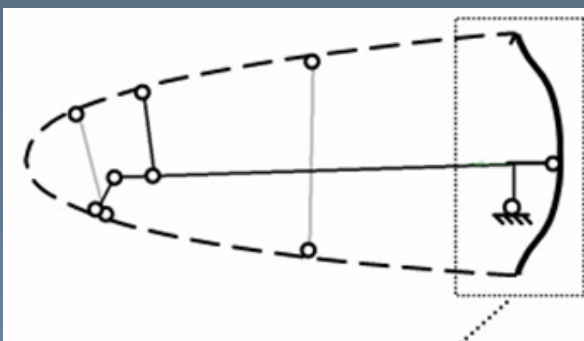
Spring features	
Shape	Thin long plate
Material	Aluminum alloy
Constraint condition	Clamped-clamped
Length (mm)	0,055
Thickness (mm)	1,8
Depth (mm)	1,0

1:6 scaled	
Re	1,00E+06
M	0,08
V inf (m/s)	29,2
Applied force (N)	110
Applied force per 1 mm span (N/mm)	5,5
Point displacement (m)	0,001905
Elastic constant (N/m per mm span)	2887

Spring features	
Shape	Thin long plate
Material	Aluminum alloy
Constraint condition	Clamped-clamped
Length (mm)	0,055
Thickness (mm)	0,8
Depth (mm)	1,0

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Spring identification for 'self adaptive' modality



Real sized	
Re	7,00E+06
M	0,075
V inf (m/s)	25,5
Applied force (N)	5000
Applied force per 1 mm span (N/mm)	31,25
Point displacement (m)	0,021547
Elastic constant (N/m per mm span)	1450

Spring features	
Shape	Thin long plate
Material	Aluminum alloy
Constraint condition	Clamped-clamped
Length (mm)	0,44
Thickness (mm)	4,8
Depth (mm)	1,0

Real sized	
Re	1,37E+06
M	0,14
V inf (m/s)	50
Applied force (N)	10000
Applied force per 1 mm span (N/mm)	62,5
Point displacement (m)	0,020355
Elastic constant (N/m per mm span)	3070

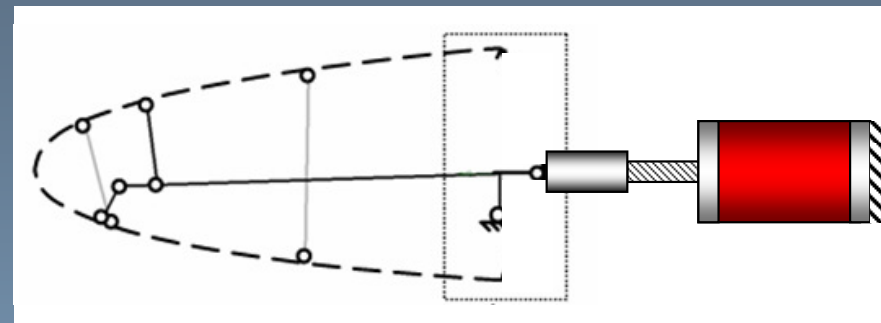
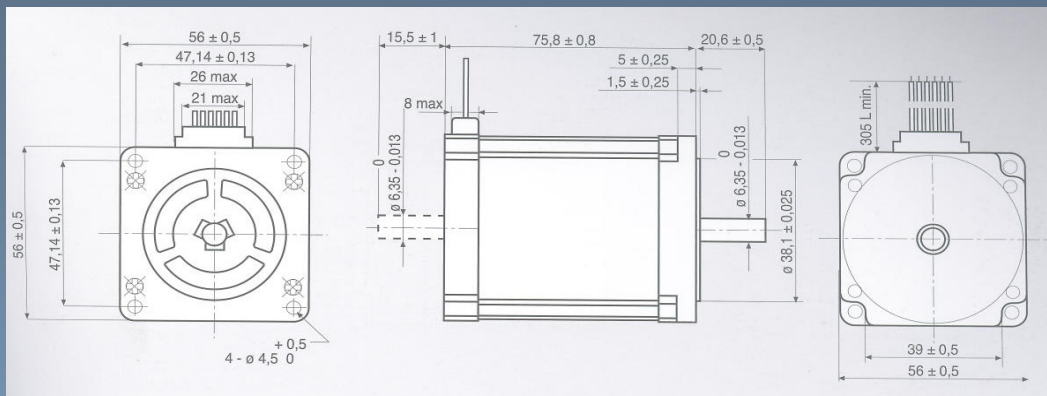
Spring features	
Shape	Thin long plate
Material	Aluminum alloy
Constraint condition	Clamped-clamped
Length (mm)	0,44
Thickness (mm)	6,2
Depth (mm)	1,0

Real sized	
Re	2,19E+06
M	0,23
V inf (m/s)	80
Applied force (N)	20000
Applied force per 1 mm span (N/mm)	125
Point displacement (m)	0,017566
Elastic constant (N/m per mm span)	7116

Spring features	
Shape	Thin long plate
Material	Aluminum alloy
Constraint condition	Clamped-clamped
Length (mm)	0,44
Thickness (mm)	8,1
Depth (mm)	1,0

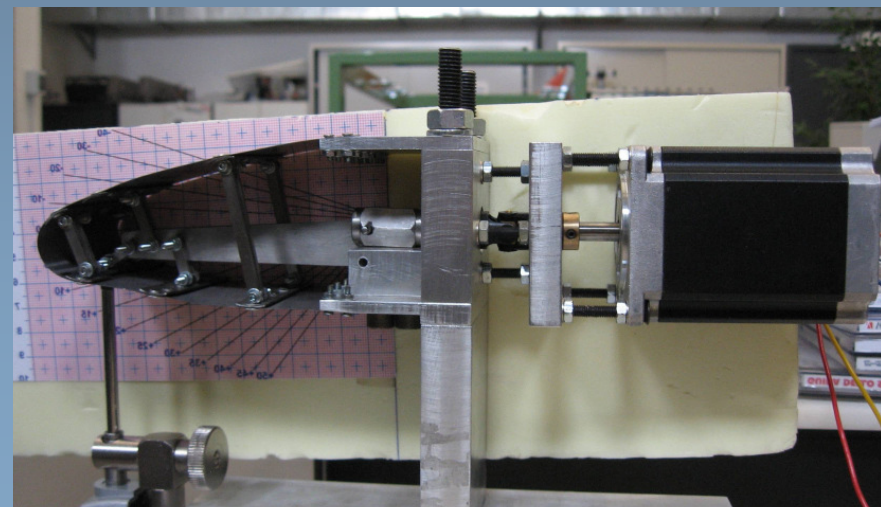
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Actuator identification for 'aided motor' modality



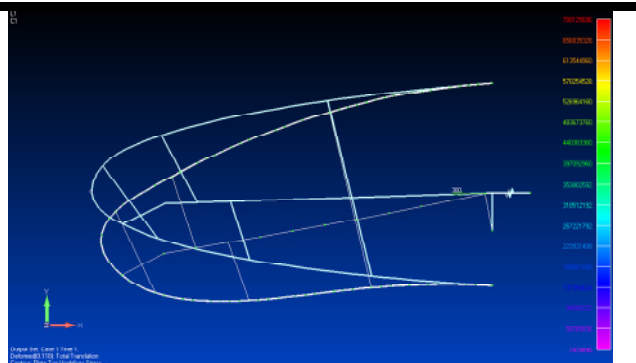
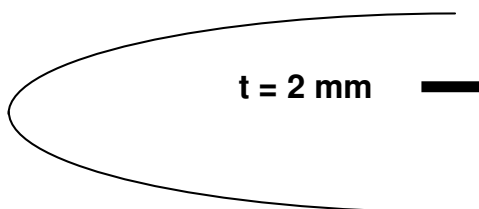
CARATTERISTICHE

MODELLO		103 - H7126 - 0140	103 - H7126 - 0740 (103 - H7126 - 0710)	103 - H7126 - 1740 (103 - H7126 - 1712)
ANGOLO DI PASSO - TOLLERANZA		1,8° ± 0,09°	1,8° ± 0,09°	1,8° ± 0,09°
CORRENTE NOMINALE BIPOLARE	(Amp)	0,75 ^(*)	2,2 ^(*)	4
CORRENTE NOMINALE UNIPOLARE	(Amp)	1	3	
RESISTENZA DI FASE A 25°	(Ohm)	8,6	0,9	0,48
INDUTTANZA DI FASE	(mH)	19	2,2	2,2
COPPIA DI TENUTA BIPOLARE	(Ncm)	165	165	165
COPPIA DI TENUTA UNIPOLARE	(Ncm)	130	130	
MOMENTO D'INERZIA ROTORE	(Kgm ² x 10 ⁻⁷)	360	360	360
ACCELERAZIONE TEORICA	(rad x sec. ⁻²)	45800	45800	45800
FORZA CONTRO ELETTRO MOTRICE	(V/Krpm)	92	31	31
PESO	(Kg)	1	1	1

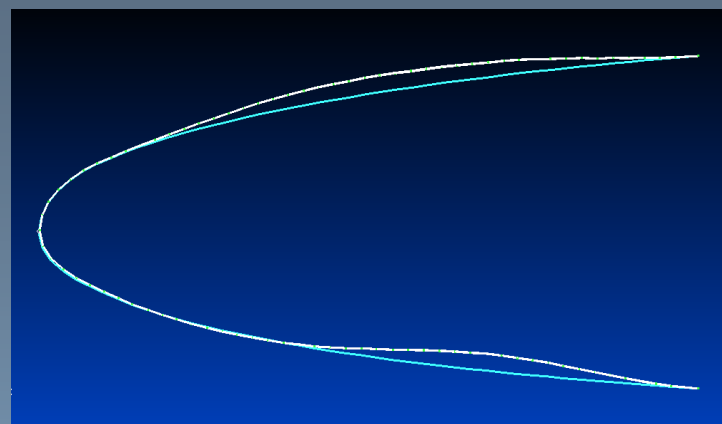


Morphing performance

Isotropic constant thickness skin



- Max vertical displacement: 3.1%
- Stress limit: 700 MPa
- Shape preserving under aerodyn. loads: 0.056% of the chord



Shape deviation (amplified)

	1:6 scaled model	Real sized model
Max displacement	16 mm (14.3% of LE zone, 3.2% of total chord)	118 mm (13.2 % of LE zone, 3.1% of total chord)
Max stress (within the skin)	700 MPa	700 MPa
LE radius increase	≈70%	≈70%



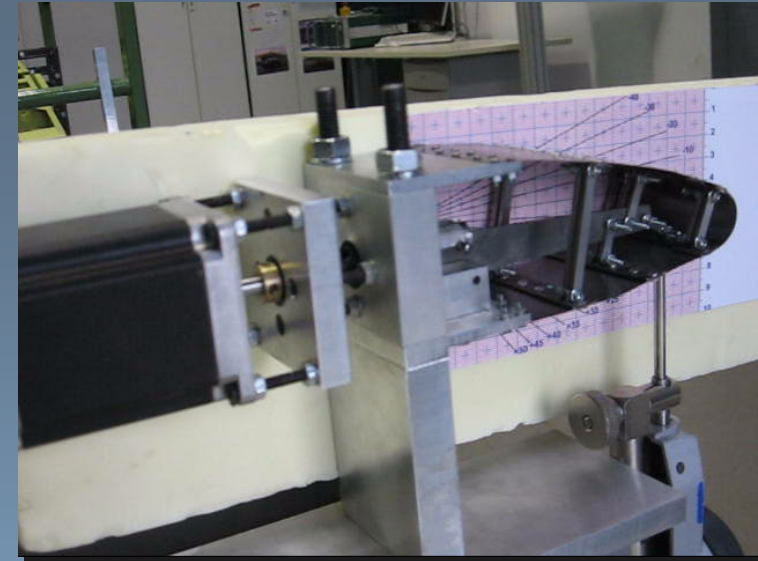
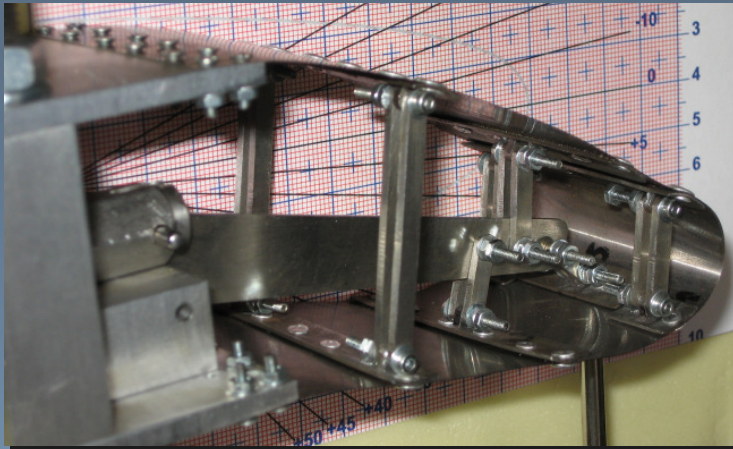
Conclusions

- A short account on traditional high lift devices, pros and cons
- The morphing approach, possible applications and expected advantages
- Motivation of the work : design of a morphing device for a smooth (laminar preserving) droop nose effect
- Description of selected architecture and definition of two working principles: '*self adaptive*' and '*motor aided*'
- Design and modeling → FE non linear approach:
 - spring identification for '*self adaptive*' modality
 - actuator identification for '*aided motor*' modality
- Discussion on device performance: vertical displacement, LE radius increase, shape keeping

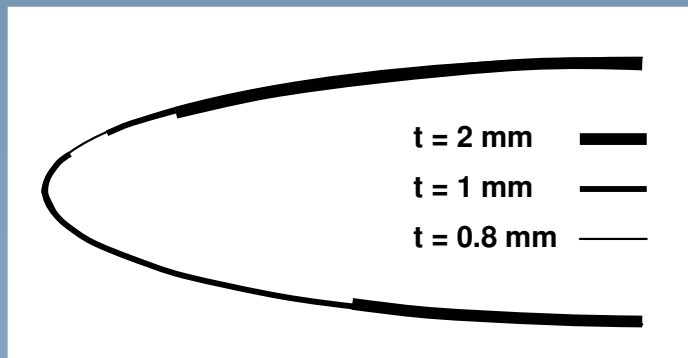


➤ 1:6 demonstrator realization and testing

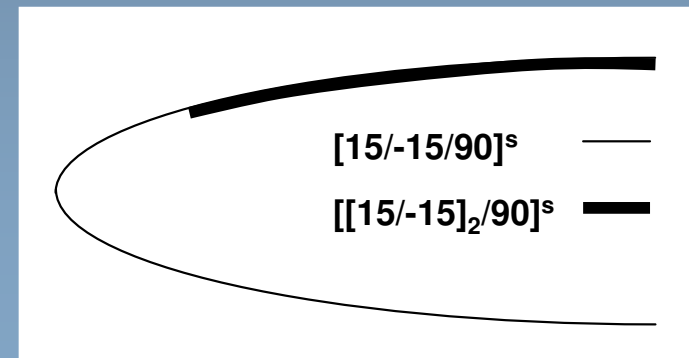
Related act. and next steps (1/2)



➤ Investigation on different skin typologies:



variable isotropic skin

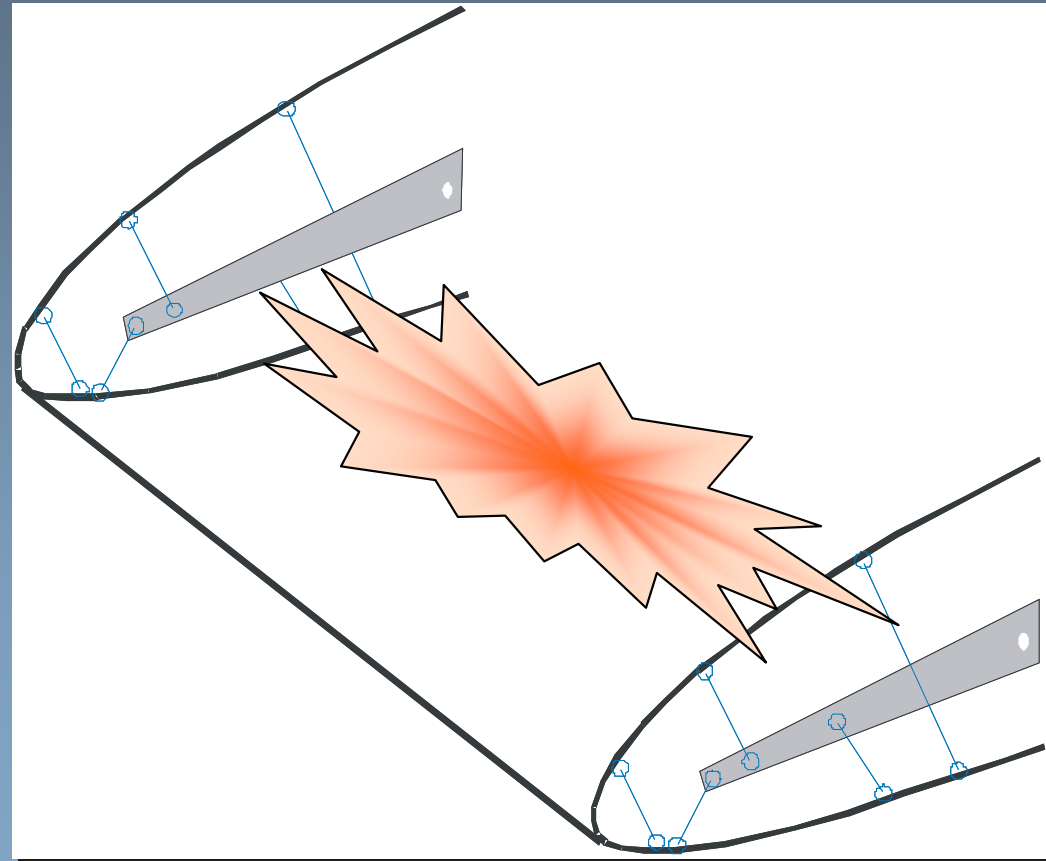


Fiber-glass laminate with variable stack sequence



➤ Real scale optimized demonstrator: design, realization and testing

- high actuation forces required towards chord-wise direction;
- better skin thickness distribution for stress alleviation and higher morphing performance
- a novel architecture modified to reduce actuation forces has been conceived and is under manufacturing;
- a dedicated skin solution has been taken into account



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Acknowledgments

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Thank you for your kind attention!

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